

# Control Design of Hard Disk Drive Concentric Self-Servo Track Writing via $H_2$ and $H_\infty$ Synthesis

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### **Abstract**

*This paper considers two novel controller synthesis methodologies using a feedforward control structure for performing concentric self-servo track writing in hard disk drives. In the first methodology, it is assumed that a conventional causal track-following controller has been designed and a non-causal feedforward controller, which utilizes the stored error signal from writing the previous track, is designed using standard  $H_\infty$  control synthesis techniques, in order to prevent the track errors from propagating and to achieve good disturbance attenuation. In the second methodology, both the track-following feedback controller and the feedforward controller are simultaneously designed via a mixed  $H_2/H_\infty$  control scheme, which involves the solution of a set of linear matrix inequalities and achieves good disturbance attenuation while preventing the propagation of track errors from the previous tracks. Simulation results confirm that the two proposed control synthesis methodologies prevent error propagation from the previously written tracks and significantly improve concentric self-servo track writing performance.*

# 1 Introduction

Modern hard disk drive (HDD) servo systems require that servo patterns, which contain track positioning information, should be embedded on the disk surface at specific locations called servo sectors. Afterwards, the servo patterns on servo sectors are utilized to provide feedback signals for HDD servo systems to position read/write magnetic head. In order to reduce track misregistration [4, 9] and increase track density [19], it is necessary to improve the precision of the servo pattern writing process.

Conventionally, servo patterns are written by costly dedicated servowriting equipment [12] external to disk drives. Self-servo track writing (SSTW) [2, 11] is an alternative method of writing servo patterns using the HDD's own reading and writing heads and servo system, in order to save the process cost and improve the servowriting quality. During SSTW, the timing and radial information are obtained from the previously written track using the read head, while timing and radial positioning servo patterns for the current track are being written using the write head. Consequentially, the external equipment is no longer needed in the servo-pattern writing and thus the servo track writing does not have to be carried out in any clean room environment. There are two most popular SSTW methodologies, spiral-based SSTW [1, 16] and concentric-based SSTW. In this paper, we just focus on the compensation scheme for concentric self-servo track writing. Specifically, the process of the concentric self-servo track writing generally involves the following steps [18]:

1. Write one or more concentric servo sector tracks using conventional servowriting methodologies. These tracks are used as the initial seed tracks, from which reference timing and radial position information is measured to write the next (adjacent) track in a bootstrap manner, and can be pre-written on the disks before the disks are assembled in the HDD.
2. Assume that the read-head to write-head position offset is equal to one track width [5]. Using the read head, collect timing and radial information from the previously written seed track and use this information to generate the position error signals to track follow the seed track, while the write head writes actual servo patterns for the current track.
3. Use the track written in Step 2) as the new seed track and go back to Step 2) until all concentric tracks are written.

However, several challenges arise with the concentric SSTW process such as the fact that radial position errors from the previous track can propagate into the currently written track. This radial positioning error propagation will lead to instability unless it is contained by guaranteeing that the magnitude for the error propagation term is sufficiently attenuated. In order to contain the error propagation, iterative learning control (ILC) and 2-Dimensional  $H_2$  control have been studied in [17] and [8] respectively. In [17], a feedforward based iterative learning control is designed in

the lifted domain assuming zero initial conditions at the beginning of each track servo writing stage and the existence of finite impulse-response (FIR) representations for the servo's sensitivity and complimentary sensitivity functions. However, we note that these assumptions are not strictly true for real HDDs. In [8], a novel 2-Dimensional  $H_2$  control synthesis technique for SSTW is formulated to satisfy a sufficient rather than a necessary condition by making some matrices be block diagonal in order to transfer the optimization to the form of linear matrix inequalities (LMI).

In this paper, we present two novel control synthesis methodologies for performing concentric self-servo track writing in hard disk drives using a feedforward control structure [17, 13]. In the first methodology, a non-causal feedforward controller, which utilizes the stored error signal [13] from writing the previous track, is designed given a pre-defined causal track-following controller. Standard  $H_\infty$  control synthesis techniques are used to avoid the propagation of track errors from the previous tracks, while achieving sufficient disturbance attenuation. In the second methodology, an analytic expression for the power spectrum density of track errors is derived and approximated. The approximate expression is subsequently used to formulate the simultaneous design of both a feedback and a feedforward controller, using a mixed  $H_2/H_\infty$  control scheme, which ensures the containment of the error propagation and the achievement of good disturbance attenuation and is solved via the solution of a set of LMIs. Neither of these techniques utilizes the simplifying assumptions in [17]. Simulation results based on the HDD benchmark problem developed in [10] show that the controllers synthesized by the proposed schemes outperform the controllers synthesized by the techniques in [17], and offer levels of performance that are comparable to the 2-dimensional  $H_2$  control technique in [8] while having a much simpler control structure.

The paper is organized as follows. Section 2 describes the non-causal feedforward control design by using standard  $H_\infty$  control. In Section 3, the analytical expression for the power spectrum density of track errors is derived. Section 4 presents the design of feedback and feedforward controllers by using a mixed  $H_2/H_\infty$  control scheme. Simulation results are provided in Section 5. Finally, conclusions are given in Section 6.

## 2 NON-CAUSAL FEEDFORWARD CONTROL DESIGN VIA $H_\infty$ CONTROL

### 2.1 Feedforward-control structure based SSTW system

Figure 1 illustrates the block diagram of the concentric self-servo track writing system with a feedforward control structure [13]. The system includes a feedforward controller  $F(z)$  and a standard track-following servo loop with the VCM plant  $P(z)$  and the feedback controller  $C(z)$ . In Fig. 1,  $i$  and  $k$  denote the track index and servo sector index respectively, while  $\Delta y_i(k)$ ,  $w_i(k)$ ,  $r_i(k)$  and  $n_i(k)$  denote the track error, windage, track runout due to disk vibrations, and measurement noise, respectively, at the position of track  $i$  and servo sector  $k$ . Similar to [15],  $d_n$ ,  $d_w$ , and  $d_r$  are

assumed to be white noises with unit variance. Moreover, since the feedforward controller  $F(z)$  utilizes the error signal  $e_{i-1}(k)$ , which can be stored when writing the previous track, and hence the entire  $e_{i-1}(k)$  sequence in  $(k)$  is available when writing the current track, a non-causal feedforward controller  $F(z)$  is feasible for the control structure in Fig. 1. Here, windage and measurement noise are modeled as white noises with the variance  $\sigma_w^2$  and  $\sigma_n^2$  respectively, while the track runout caused by disk vibrations is modeled as a color noise generated by feeding a white noise  $d_r$  input to the filter  $G_r(z)$ .

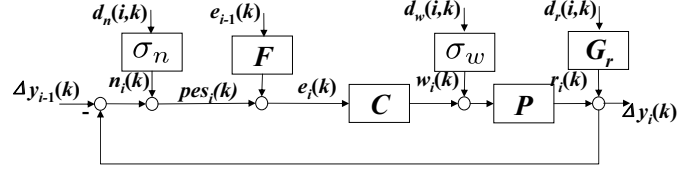


Figure 1: Feedforward control structure based SSTW system

Based on the block diagram in Fig. 1, we can get the following recursive expression for track errors:

$$\begin{aligned} \Delta y_i(k) = & G_1(z)\Delta y_{i-1}(k) + T(z)n_i(k) \\ & + S(z)d_i(k) - S(z)F(z)d_{i-1}(k) \end{aligned} \quad (1)$$

where  $T(z) = \frac{P(z)C(z)}{1+P(z)C(z)}$ ,  $S(z) = \frac{1}{1+P(z)C(z)}$ ,  $d_i(k) = P(z)w_i(k) + r_i(k)$ , and  $G_1(z) = \frac{P(z)C(z)+F(z)}{1+P(z)C(z)}$ .

Notice that  $G_1(z)$  becomes the key transfer function relating the previous and the current track errors.

## 2.2 Non-causal feedforward control design

Like the iterative learning control in [17] and [14], a feedback controller  $C(z)$  for track following is firstly designed to achieve good disturbance attenuation. Here,  $C(z)$  is designed as an optimal  $H_2$  controller. In order to contain the error propagation, the designed controllers must satisfy  $\|G_1(e^{j\omega})\|_\infty < 1$ . Furthermore, in order to make the error propagation converge as quickly as possible, we want  $\|G_1(e^{j\omega})\|_\infty$  to be sufficiently small. From (1), we learn that the current track error is also affected by the disturbances from the previous track. In order not to degrade the disturbance attenuation performance of the track-following controller  $C(z)$ , the magnitude of the filter  $F(z)$  needs to also be sufficiently small. In all, the feedforward control  $F(z)$  must be designed to achieve the following target:

$$\begin{cases} \|G_1(z)\|_\infty : \text{sufficiently small and less than 1} \\ \|F(z)\|_\infty : \text{sufficiently small} \end{cases} \quad (2)$$

As a consequence, we consider the following optimization:

$$\min_{F(z)} \left\| \begin{bmatrix} G_1(z) & w_{t1}F(z) \end{bmatrix} \right\|_{\infty} \quad (3)$$

where  $w_{t1}$  is a weighting value to be tuned to achieve the target in (2). The optimization in (3) is a standard  $H_{\infty}$  control problem, which can be easily solved, as shown later in this section. However, the solution to (3) can only produce a causal compensator  $F(z)$  [7]. Obviously, a smaller objective value may be achieved if  $F(z)$  is allowed to be non-causal.

In order to design a non-causal filter  $F(z)$ , we consider the following facts:

$$\begin{aligned} \left\| \begin{bmatrix} G_1(z) & w_{t1}F(z) \end{bmatrix} \right\|_{\infty} &= \left\| \begin{bmatrix} z^{-n_d}G_1(z) & w_{t1}z^{-n_d}F(z) \end{bmatrix} \right\|_{\infty} \\ &= \left\| \begin{bmatrix} \frac{z^{-n_d}P(z)C(z)+\tilde{F}(z)}{1+P(z)C(z)} & w_{t1}\tilde{F}(z) \end{bmatrix} \right\|_{\infty} \end{aligned} \quad (4)$$

where  $F(z) = z^{n_d}\tilde{F}(z)$  and  $n_d$  is a positive integer. Thus, the optimization in (3) can be transformed into the following optimization:

$$\min_{\tilde{F}(z)} \left\| \begin{bmatrix} \frac{z^{-n_d}P(z)C(z)+\tilde{F}(z)}{1+P(z)C(z)} & w_{t1}\tilde{F}(z) \end{bmatrix} \right\|_{\infty}. \quad (5)$$

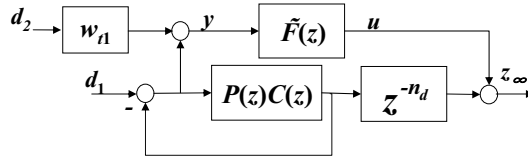


Figure 2: Block diagram for the interpretation of  $H_{\infty}$  norm in (5)

By considering the block diagram in Fig. 2, we have  $T_{z_{\infty} \leftarrow d_{\infty}} = \begin{bmatrix} \frac{z^{-n_d}P(z)C(z)+\tilde{F}(z)}{1+P(z)C(z)} & w_{t1}\tilde{F}(z) \end{bmatrix}$ , where  $d_{\infty} = \begin{bmatrix} d_1 & d_2 \end{bmatrix}^T$ . Note that the symbol  $T_{A \leftarrow B}$  represents the transfer function from signal  $B$  to signal  $A$ . Thus, the optimization in (5) can be interpreted as an  $H_{\infty}$  control problem for the linear fractional transformation (LFT) in Fig. 3 to minimize  $\|T_{z_{\infty} \leftarrow d_{\infty}}\|_{\infty}$ . Here,  $G(z)$  is the transfer function matrix from  $\begin{bmatrix} d_{\infty}^T & u \end{bmatrix}^T$  to  $\begin{bmatrix} z_{\infty}^T & y \end{bmatrix}^T$  as shown in Fig. 2.

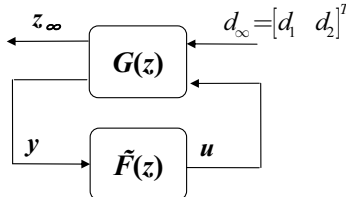


Figure 3: LFT for the  $H_{\infty}$  control design problem

Then, the standard  $H_\infty$  control synthesis technique can be applied to the control problem as shown in Fig. 3 to generate the controller  $\tilde{F}(z)$ . Once a causal  $\tilde{F}(z)$  is designed, a non-causal feedforward controller can be constructed from  $F(z) = z^{n_d} \tilde{F}(z)$ .

### 3 Track error analysis for the feedforward control based SSTW

#### 3.1 Power spectrum density of track errors

In order to investigate the relationship between the current track error and disturbances from the previously written tracks, we assume that the servo patterns on each track are written after the system reaches its steady state. Then, based on the recursive form of track errors in (1), we have the following complete expression for track errors.

$$\begin{aligned}
\Delta y_i(k) &= G_1^i \Delta y_0(k) + \sum_{l=1}^i G_1^{i-l} T n_l(k) + \sum_{l=1}^i G_1^{i-l} S d_l(k) \\
&\quad - \sum_{l=1}^{i-1} G_1^{i-1-l} S F d_l(k) \\
&= \sum_{l=1}^{i-1} G_1^{i-1-l} T [(T + S F) n_l(k) - S(1 - F) d_l(k)] \\
&\quad + G_1^i \Delta y_0(k) + T n_i(k) + S d_i(k) \\
&= \sum_{l=1}^{i-1} G_1^{i-1-l} T [G_1 n_l(k) - S(1 - F) d_l(k)] \\
&\quad + G_1^i \Delta y_0(k) + T n_i(k) + S d_i(k). \tag{6}
\end{aligned}$$

Furthermore, we assume that the seed track error  $\Delta y_0$ , measurement noises and disturbances are uncorrelated with each other and the track error on the seed track has a power spectrum density  $\Phi_{\Delta y_0 \Delta y_0}$ . Moreover, measurement noises on different tracks are uncorrelated and have the same variance  $\sigma_n^2$ , while disturbances on different tracks are also uncorrelated and have the same power spectrum density  $\Phi_{dd}(e^{j\omega})$ . With these assumptions, we can get the following power spectrum density for the track error on track  $i$ :

$$\begin{aligned}
\Phi_{\Delta y_i \Delta y_i}(e^{j\omega}) &= \sum_{l=1}^{i-1} |G_1|^{2(i-l-1)} |T|^2 (|G_1|^2 \sigma_n^2 + |S|^2 |1 - F|^2 \\
&\quad \cdot \Phi_{dd}) + |G_1|^{2i} \Phi_{\Delta y_0 \Delta y_0} + |T|^2 \sigma_n^2 + |S|^2 \Phi_{dd}. \tag{7}
\end{aligned}$$

When the track index  $i$  is quite large,  $|G_1(e^{j\omega})|^{2i}$  will be closed to zero, since  $|G_1(e^{j\omega})| < 1$ . Then, for the large track index  $i$ , we have:

$$\begin{aligned} \Phi_{\Delta y_i \Delta y_i}(e^{j\omega}) &= \frac{|T|^2}{1-|G_1|^2} [\sigma_n^2 + |\hat{F}|^2 \Phi_{dd}(e^{j\omega})] \\ &\quad + |S|^2 \Phi_{dd}(e^{j\omega}). \end{aligned} \quad (8)$$

In order to conveniently synthesize the mixed  $H_2/H_\infty$  control, which will be discussed in Section 4, we utilize the parameterization  $G_1(z) = 1 + \hat{F}(z)$  and  $\hat{F}(z) = S(z)(F(z) - 1)$ .

### 3.2 Discussion

From (8), we note that, in order to reduce track errors, not only a good track-following feedback controller is necessary, but also both  $|G_1(e^{j\omega})|$  and  $|\hat{F}(e^{j\omega})|$  should be sufficiently small. However,  $|G_1(e^{j\omega})|$  and  $|\hat{F}(e^{j\omega})|$  can not be closed to zero at the same time, since  $G_1(z) = 1 + \hat{F}(z)$ . Intuitively, in order to accomplish a good tracking performance, the  $H_2$  norm of the transfer functions from disturbances to track errors must be minimized and simultaneously an appropriately small  $\|G_1(z)\|_\infty$  must be guaranteed. This idea turns out to be a mixed  $H_2/H_\infty$  control problem, which will be discussed in next section.

## 4 The design of feedback and feedforward control by using mixed $H_2/H_\infty$ synthesis

### 4.1 Problem formulation

As discussed in Section 3, (8) educs the idea of a mixed  $H_2/H_\infty$  control design in order to achieve a good tracking performance. Let's rewrite (8) as:

$$\begin{aligned} \Phi_{\Delta y_i \Delta y_i}(e^{j\omega}) &= |T|^2 \left( \frac{\sigma_n^2}{1-|G_1|^2} \right) + |\hat{F}|^2 \left( \frac{|T|^2}{1-|G_1|^2} \times \right. \\ &\quad \left. \Phi_{dd}(e^{j\omega}) \right) + |S|^2 \Phi_{dd}(e^{j\omega}). \end{aligned} \quad (9)$$

Clearly, (9) demonstrates that the track error can be considered as the output of the system  $\bar{G}_2(z) = \begin{bmatrix} T(z) & \hat{F}(z) & S(z) \end{bmatrix}$

with the input of  $\begin{bmatrix} \frac{n_i}{(1-|G_1|^2)^{1/2}} & \frac{T}{(1-|G_1|^2)^{1/2}} \tilde{d}_i & d_i \end{bmatrix}^T$ . Here,  $\tilde{d}_i$ 's are artificial disturbances, which are uncorrelated with  $n_i$  and  $d_i$  and have the same power spectrum density as  $d_i$ . Notice that the artificial disturbances  $\tilde{d}_i$ 's are just introduced to represent the second component of the track error power spectrum density in (9).



Since the weighting functions  $\frac{1}{(1-|G_1|^2)^{1/2}}$  and  $\frac{T}{(1-|G_1|^2)^{1/2}}$  for  $n_i$  and  $\tilde{d}_i$  are not affine in  $G_1(z)$  and  $T(z)$ , the two weighting functions are replaced by two weighting values  $w_{t2}$  and  $w_{t3}$  respectively, in order to conveniently construct a linear system to represent the transfer function matrix from the input  $\begin{bmatrix} n_i & \tilde{d}_i & d_i \end{bmatrix}^T$  to  $\Delta y_i$ . Such substitution is further validated by the fact that the magnitude frequency responses of both  $G_1$  and  $T$  are expected to be flat at low and middle frequencies [17].

By considering the system denoted in Fig. 4 where  $d_w$ ,  $d_r$ ,  $d_n$ ,  $\tilde{d}_w$ , and  $\tilde{d}_r$  are assumed to be uncorrelated white noises, we obtain the following expression for the power spectrum density of  $z_2$ :

$$\Phi_{z_2 z_2}(e^{j\omega}) = |T|^2 w_{t2}^2 \sigma_n^2 + |\hat{F}|^2 w_{t3}^2 \Phi_{dd}(e^{j\omega}) + |S|^2 \Phi_{dd}(e^{j\omega}). \quad (10)$$

Obviously, the power spectrum density of  $z_2$  is similar to that of  $\Delta y_i$  except the replacement of the weighting functions in (9) with the corresponding weighting values in (10). Thus, with appropriate weighting values  $w_{t2}$  and  $w_{t3}$ ,  $\Phi_{\Delta y_i \Delta y_i}(e^{j\omega})$  can be approximated by  $\Phi_{z_2 z_2}(e^{j\omega})$ . As shown in Fig. 4, let  $G_2(z) = \begin{bmatrix} T \\ z_2 \leftarrow \begin{bmatrix} d_w & d_r & d_n \end{bmatrix}^T & w_{t3} \hat{F} * T \\ \tilde{d} \leftarrow \begin{bmatrix} \tilde{d}_w & \tilde{d}_r \end{bmatrix}^T \end{bmatrix}$  denote the transfer function matrix from  $\begin{bmatrix} d_w & d_r & d_n & \tilde{d}_w & \tilde{d}_r \end{bmatrix}^T$  to  $z_2$ . Therefore, in order to accomplish good track-

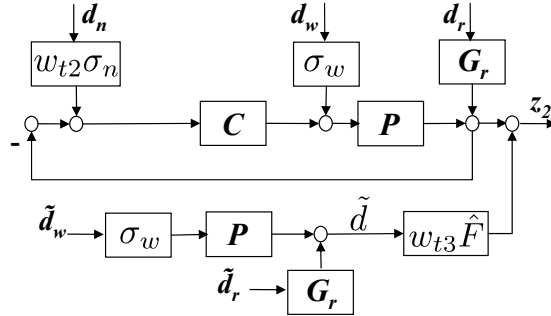


Figure 4: Block diagram for the interpretation of  $G_2(z)$

ing error performance, we consider the mixed  $H_2/H_\infty$  optimization problem

$$\begin{aligned} \min_{C(z), F(z)} \quad & \|G_2(z)\|_2 \\ \text{s.t.} \quad & \|G_1\|_\infty < \gamma_0 < 1 \end{aligned} \quad (11)$$

to design  $C(z)$  and  $F(z)$  simultaneously. Here,  $\gamma_0$  is a given constant to guarantee good convergence for the track error propagation and a good attenuation for the disturbances from the previously written tracks.



$$\hat{F} * \begin{bmatrix} 0 & 0 & 0 & T \\ & & & z_2 \leftarrow [\tilde{d}_w \quad \tilde{d}_r]^T \end{bmatrix} \sim \left[ \begin{array}{c|c} \bar{A}_f & \bar{B}_f \\ \hline \bar{C}_f & \bar{D}_f \end{array} \right]$$

$$= \left[ \begin{array}{cc|c} \bar{A}_d & 0 & B_d \\ \hline B_{\hat{F}}C_d & A_{\hat{F}} & B_{\hat{F}}D_d \\ \hline D_{\hat{F}}C_d & C_{\hat{F}} & D_{\hat{F}}D_d \end{array} \right] \quad (17)$$

$$G_2 \sim \left[ \begin{array}{c|c} \bar{A}_{cl2} & \bar{B}_{cl2} \\ \hline \bar{C}_{cl2} & \bar{D}_{cl2} \end{array} \right] = \left[ \begin{array}{cc|c} \bar{A}_{cl2} & 0 & B_{cl2} \\ \hline 0 & \bar{A}_f & \bar{B}_f \\ \hline C_{cl2} & \bar{C}_f & D_{cl2} + \bar{D}_f \end{array} \right]. \quad (18)$$

Then the optimization in (12) can be synthesized as the following optimization [3]:

$$\min_{A_c, B_c, C_c, D_c, C_{\hat{F}}, D_{\hat{F}}} \text{trace}(W)$$

$$\text{s.t.} \quad \left[ \begin{array}{ccc} W & \bar{C}_{cl2} & \bar{D}_{cl2} \\ * & X_2 & 0 \\ * & * & I \end{array} \right] \succ 0 \quad (19)$$

$$\left[ \begin{array}{ccc} X_2 & X_2 \bar{A}_{cl2} & P_2 \bar{B}_{cl2} \\ * & X_2 & 0 \\ * & * & I \end{array} \right] \succ 0 \quad (20)$$

$$\left[ \begin{array}{cccc} X_1 & X_1 A_{\hat{F}} & X_1 B_{\hat{F}} & 0 \\ * & X_1 & 0 & C_{\hat{F}}^T \\ * & * & I & 1 + D_{\hat{F}}^T \\ * & * & * & \gamma_0^2 I \end{array} \right] \succ 0 \quad (21)$$

where the symbol "\*" denotes the transpose of the corresponding element at its transposed position. Since both  $X_1$  and  $X_2$  are coupled with  $A_{\hat{F}}$  and  $B_{\hat{F}}$  in (21) and (20) respectively, the filter  $\hat{F}(z)$  is chosen as an FIR filter, which means that  $A_{\hat{F}}$  and  $B_{\hat{F}}$  are known and thus (21) becomes an LMI. Moreover, in order to recover the convexity of (19) and (20) by an appropriate nonlinear transformation [6], the matrix  $X_2$  is chosen as a block diagonal matrix, i.e.,  $X_2 = \text{diag}\{X_{22}, X_{ff}\}$ . As a result, the optimization involving (19), (20) and (21) is a convex optimization, which can be easily solved. After synthesizing  $\hat{F}(z)$ , we can reconstruct the feedforward control by

$$F(z) = 1 + S^{-1}(z)\hat{F}(z) = 1 + (1 + P(z)C(z))\hat{F}(z).$$

## 5 Simulation Study

In order to evaluate the concentric SSTW design methodologies presented in this paper, they will be tested through a simulation study that utilizes the benchmark model developed by the IEEJapan technical committee on Nano-Scale Servo (NSS) system [10]. This model was also utilized to test the concentric SSTW design scheme presented in [17]. This benchmark model was originally developed to test track-following servos and must be modified to test servo systems for self-servo track writing control. For the simulated drive, the servo sector number  $N$  is equal to 220 and the disk rotation speed is 7200 RPM. Thus, the sampling frequency for this drive is  $f_s = 220 * 7200 / 60 = 26400$  Hz.

### 5.1 Weighting value determination

Before using the presented two control synthesis methodologies, we have to determine the corresponding weighting values  $w_{t1}$ ,  $w_{t2}$ , and  $w_{t3}$ . For the technique presented in Section 2, since the track error propagation term  $G_1$  may result in instability (if  $\|G_1(z)\|_\infty > 1$ ), it is reasonable to choose a relatively small  $w_{t1}$ . Intuitively, the selection of  $w_{t1} < 1$  is desirable to emphasize  $G_1$ . For the technique presented in Section 4,  $\gamma_0$  must be less than 1 and is required to be closed to 1 so that the obtained  $H_2$  norm cost for track errors in (11) is not too conservative according to the  $H_\infty$  norm constraint. In addition,  $w_{t2}$  and  $w_{t3}$  are utilized to approximate  $\frac{1}{(1-|G_1|^2)^{1/2}}$  and  $\frac{T}{(1-|G_1|^2)^{1/2}}$  respectively. As mentioned in Section 4, the magnitude of  $T$  and  $G_1$  are flat at the low and middle frequencies and thus it is desirable to determine  $w_{t2}$  and  $w_{t3}$  using the DC gains of  $T$  and  $G_1$ . It is well known that in order to attenuate low-frequency disturbances,  $T = \frac{PC}{1+PC}$  is usually designed to have a unit DC gain. Thus,  $w_{t2}$  and  $w_{t3}$  can be roughly selected by

$$w_{t2} = w_{t3} = \frac{1}{\left(1 - (\text{Expected DC Gain of } G_1)^2\right)^{1/2}}.$$

### 5.2 Control design results

An optimal  $H_2$  track following compensator  $C(z)$  was first synthesized and then a non-causal feedforward compensator  $F(z)$  was designed using the  $H_\infty$  control design methodology presented in Section 2, with the weighting value  $w_{t1} = 0.16$  and  $n_d = 7$ . The designed control system achieves  $\|G_1(z)\|_\infty = 0.9781 < 1$  and  $\|F(z)\|_\infty = 1.3633$ . The corresponding frequency response plots for the designed  $F(z)$ ,  $\frac{P(z)C(z)+F(z)}{1+P(z)C(z)}$ ,  $\frac{1}{1+P(z)C(z)}$ , and  $\frac{F(z)}{1+P(z)C(z)}$  are shown in Fig. 5.

Subsequently, a feedforward compensator  $F(z)$  constructed from the FIR filter  $\hat{F}(z)$  and a feedback compensator  $C(z)$  were simultaneously designed using the mixed  $H_2/H_\infty$  control synthesis methodology in Section 4. The designed control system achieves  $\|G_1(z)\|_\infty = 0.9737$  with the tuning parameters  $w_{t2} = 4$ ,  $w_{t3} = 4$  and  $\gamma_0 = 0.98$ . The frequency

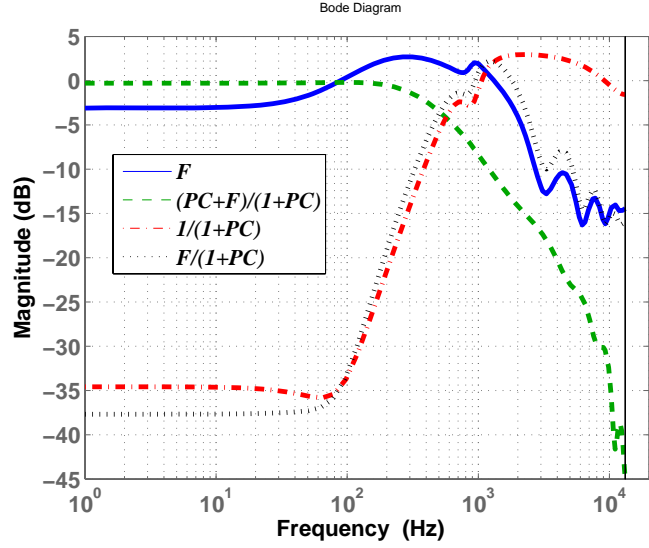


Figure 5: Frequency responses for the non-causal feedforward control design via  $H_\infty$  in Section 2

response plots for the resulting controllers are shown in Fig. 6.

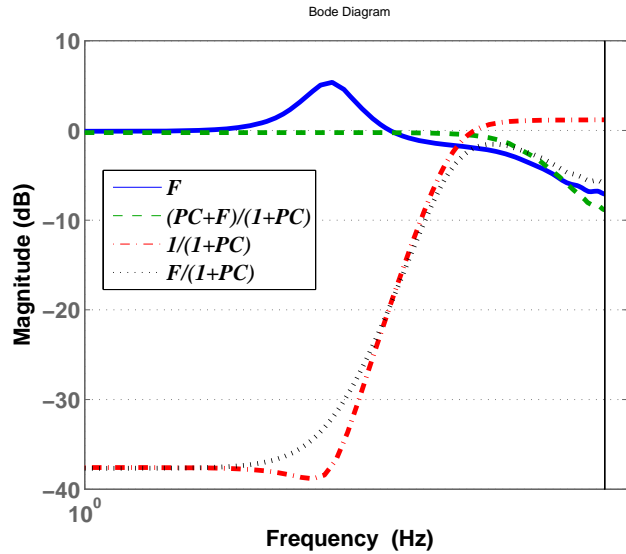


Figure 6: Frequency responses for the feedback and feedforward control designs using the mixed  $H_2/H_\infty$  synthesis methodology in Section 4.

### 5.3 Time-domain simulation results

For the benchmark problem in [10], the modeled sensor noise has a sigma value of 1.5% of track pitch; that of the track runout due to disk vibrations is 1.7% of track pitch; the contribution of the windage at PES has a sigma value of 12.2% track width. The track error for the seed track is assumed to be a sigma value of 14% track width. In the simulation, a total of 5000 servo tracks data was collected. In order to interpret the simulated results better, we also

provide the time-domain simulation results for the 2-Dimensional  $H_2$  SSTW synthesis technique presented in [8]. The sigma values of the first 5000 self-servo written tracks for the proposed two methods in this paper and for the 2-D  $H_2$  system are depicted in Fig. 7. Obviously, the track error propagation is well contained for all the three design methodologies.

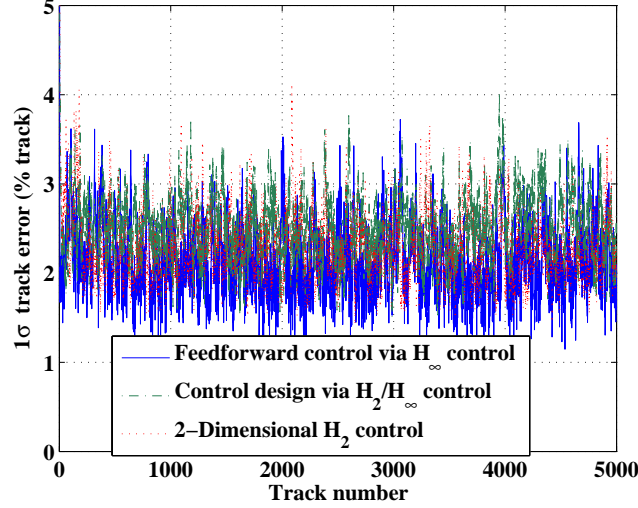


Figure 7: Time domain simulation results for track errors. Since the performance of the control design via  $H_2/H_\infty$  is closed to that of 2-D  $H_2$  control, the green dashed line is almost covered by the red dotted line.

Meanwhile, by considering the relatively large variance of the seed track, we are also interested in checking how fast the transition response caused by the seed track can converge. The zoomed in figure for the transition response is illustrated in Fig. 8. The results demonstrate that the effect of the bad seed track on the subsequently written tracks by the proposed controllers disappears very quickly. Specifically, the simulation results show that the transition responses have disappeared after about 15 tracks.

We now consider another common performance index called AC squeeze in order to quantify the quality of written tracks. The AC squeeze for track  $i$  is defined as:

$$\text{ACsqueeze}_i = \min_{k \in [0, N-1]} \{1 + \Delta y_i(k) - \Delta y_{i-1}(k)\} \quad (22)$$

where track errors  $\Delta y_i(k)$  and  $\Delta y_{i-1}(k)$  are normalized by the track width. When the AC squeeze is too small, two adjacent tracks with narrow track spacing may interfere with each other and cause data corruption. The ideal value of AC squeeze is 1 track width, which means the adjacent tracks are perfectly parallel to each other. The AC squeeze values for the simulated self-servo written tracks are shown in Fig. 9. Moreover, the resulting average values of  $\sigma(\Delta y_{i-1}(k))$  and  $\text{ACsqueeze}_i$  are presented in Table 1. Note that the non-causal feedforward control design through standard  $H_\infty$  control achieves the best performance for track errors, while the feedback and feedforward control designs

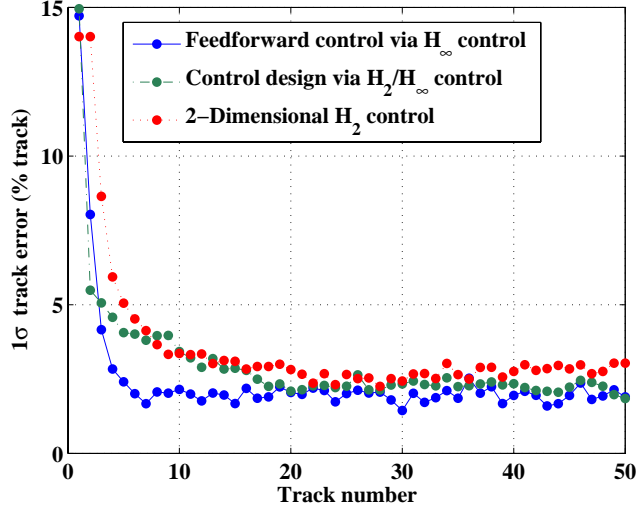


Figure 8: Zoomed in Fig. 7 to check the transition response caused by the seed track.

by using the mixed  $H_2/H_\infty$  control accomplish the best AC squeeze.

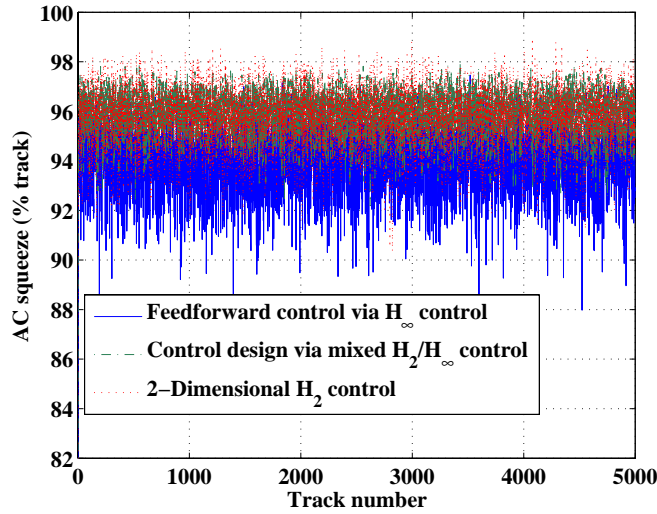


Figure 9: Time domain simulation results for AC Squeeze. Since the performance of the control design via  $H_2/H_\infty$  is closed to that of 2-D  $H_2$  control, the green dashed line is almost covered by the red dotted line.

In order to provide the better evaluation for our proposed control synthesis techniques, the simulation results reported in [17] by using the iterative learning control in lifted domain are also listed in Table 1. Obviously, the two proposed control design methodologies are able to improve both track errors and AC squeeze compared to the ILC technique. Meanwhile, the two proposed control design techniques offer the comparable performances to the 2-Dimensional  $H_2$  control technique in [8] while having a much simpler control structure.

Table 1: Simulation Results

	Non-causal feedforward control using $H_\infty$ control	Control designs using mixed $H_2/H_\infty$ control	2-D $H_2$ control	ILC in lifted domain [21]
Average of $1\sigma$ track error (% track)	2.11	2.50	2.27	2.88
Average of AC squeeze (% track)	94.0	96.0	95.8	88

## 6 Conclusion

This paper discussed two novel controller synthesis methodologies for performing concentric self-servo track writing in hard disk drives using a feedforward control structure. In the first methodology, it is assumed that a conventional causal track-following controller has been designed and then a non-causal feedforward controller, which utilizes the stored error signal from writing the previous track, is designed based on standard  $H_\infty$  control synthesis techniques. The designed controllers were used to prevent the track errors from previous tracks from propagating and to achieve good disturbance attenuation. In the second methodology, an analytic expression for the power spectrum density of track errors was approximately derived. The expression was subsequently used to formulate the simultaneous design of both feedback and feedforward controllers, using a mixed  $H_2/H_\infty$  control scheme, which ensures the containment of the error propagation and the achievement of good disturbance attenuation and was solved via the solution of a set of LMIs. Neither of these techniques utilizes the simplifying assumptions in [17]. Simulation results using the HDD benchmark problem developed in [10] showed that the controllers synthesized using the proposed schemes outperform the controllers synthesized by the techniques in [17], and offer levels of performances that are comparable with the 2-dimensional  $H_2$  control technique in [8] while having a simpler structure. Moreover, the track error propagation converges after about 15 tracks despite the seed track having a large track error.

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